

5.1 Linear Models: Initial Value Problems

Study

$$a \frac{d^2 y}{dt^2} + b \frac{dy}{dt} + cy = g(t)$$

$$y(0) = y_0$$

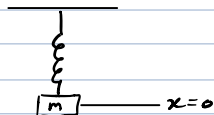
$$y'(0) = y_1$$

Driving function: $g(t)$

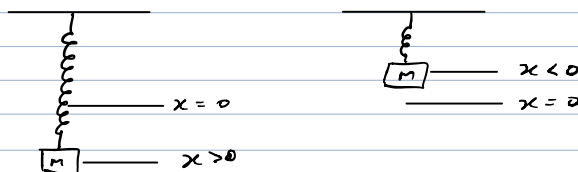
Response: a solution $y(t)$.

5.1.1 Spring/Mass Systems: Free Undamped Motion

Equilibrium:



In Motion:



DE of Free undamped Motion:

$$m \frac{d^2 x}{dt^2} = -Kx \iff \frac{d^2 x}{dt^2} + \omega^2 x = 0, \quad \omega^2 = K/m$$

m = mass

K = *spring constant* = $\frac{\text{weight}}{\text{how far mass stretches spring}}$

Weight = $w = g \cdot m$, $g = \begin{cases} 32 \text{ ft/s}^2 \\ 9.8 \text{ m/s}^2 \end{cases}$ depends on units used

The equation describes *simple harmonic motion / free undamped motion*

Free: no driving force

undamped: no resisting force against the motion

Equation of Motion:

$$x'' + \omega^2 x = 0 \rightarrow m^2 + \omega^2 = 0 \rightarrow m = \pm i\omega \rightarrow x(t) = C_1 \cos(\omega t) + C_2 \sin(\omega t)$$

Period of motion: $T = 2\pi/\omega$ = time (in sec) it takes mass to complete one cycle of motion

Extreme displacement: maximal displacement from equilibrium

Ex

mass of 2 lb stretches a spring $\frac{1}{2}$ ft. At $t=0$, the mass is released from a point $\frac{2}{3}$ ft below equilibrium position with upward velocity of $\frac{4}{3}$ ft/s. Find equation of motion.

Want to solve

$$x'' + \omega^2 x = 0, \quad \omega^2 = k/m$$
$$x(0) = -\frac{2}{3}$$
$$x'(0) = \frac{4}{3}$$

m: $w = 2 = mg \rightarrow m = \frac{w}{g} = \frac{2}{32} = \frac{1}{16}$ slug

K: $k = \frac{\text{weight}}{\text{how far mass stretches spring}} = \frac{2}{1/2} = 4 \text{ lb/ft}$

$$\therefore \omega^2 = k/m = 4/16 = 64 \rightarrow \omega = 8$$

$\therefore$ equation of motion is

$$x(t) = c_1 \cos 8t + c_2 \sin 8t$$

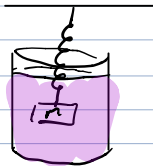
IVP: $x(0) = 2/3, \quad x'(0) = -4/3 \rightarrow c_1 = \frac{2}{3}, \quad c_2 = -\frac{1}{6}$

$\therefore x(t) = \frac{2}{3} \cos 8t - \frac{1}{6} \sin 8t$ is the equation of motion

5.1.2 Spring/Mass Systems: Free Damped Motion

Now assume mass experiences resisting (damping) force from surrounding medium.

E.g.:



Assume: damping force is proportional to instant. velocity:

$$F_{\text{damp}} = \beta x'$$

DE of Free Damped Motion :

$$m x'' = -kx - \beta x' \iff x'' + 2\lambda x' + \omega^2 x = 0$$

$\beta > 0$: *damping constant*

$$2\lambda = \beta/m$$

$$\omega^2 = k/m$$

Aux. Eq $m^2 + 2\lambda m + \omega^2 = 0 \rightarrow m_1 = -\lambda + \sqrt{\lambda^2 - \omega^2}, m_2 = -\lambda - \sqrt{\lambda^2 - \omega^2}$

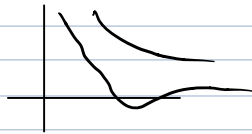
Three Cases for Damping:

Case I: $\lambda^2 - \omega^2 > 0$ ($\beta \gg k$)

Overdamped motion : $x(t) = c_1 e^{(-\lambda + \sqrt{\lambda^2 - \omega^2})t} + c_2 e^{(-\lambda - \sqrt{\lambda^2 - \omega^2})t}$

$$= \underbrace{e^{-\lambda t}}_{\text{damping}} \left(\underbrace{c_1 e^{\sqrt{\lambda^2 - \omega^2} t} + c_2 e^{-\sqrt{\lambda^2 - \omega^2} t}}_{\text{exponential behavior}} \right)$$

Eq. describes non-oscillatory motion :

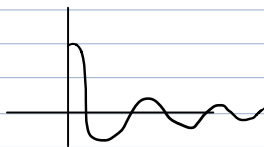


Case II: $\lambda^2 - \omega^2 < 0$ ($\beta \ll k$)

Underdamped Motion : $m_1 = -\lambda + i\sqrt{\omega^2 - \lambda^2}, m_2 = -\lambda - i\sqrt{\omega^2 - \lambda^2}$

$$x(t) = \underbrace{e^{-\lambda t}}_{\text{damping}} \left(\underbrace{c_1 \cos(\sqrt{\omega^2 - \lambda^2} t) + c_2 \sin(\sqrt{\omega^2 - \lambda^2} t)}_{\text{oscillation}} \right)$$

Eq. describes damped oscillation :



Case III: $\lambda^2 - \omega^2 = 0$ ($\frac{\beta^2}{4} = k$)

Critically Damped motion : $m_1 = m_2 = -\lambda$

$$x(t) = \underbrace{e^{-\lambda t}}_{\text{damping}} \left(\underbrace{c_1 + c_2 t}_{\text{critical motion}} \right)$$

Critical: $\begin{cases} \text{increase } \beta \rightarrow \lambda^2 - \omega^2 > 0 \rightarrow \text{overdamped} \\ \text{decrease } \beta \rightarrow \lambda^2 - \omega^2 < 0 \rightarrow \text{underdamped} \end{cases}$

Ex

mass of 8 lb stretches spring 2 ft. Assume damping force is 2 times instant. vel.
mass is released from equilibrium position with an upward velocity of 3 ft/s.
Find eq. of motion and extreme displacement.

Want to solve

$$x'' + 2\gamma x' + \omega^2 x = 0$$

$$x(0) = 0$$

$$x'(0) = -3$$

$$m = \frac{W}{g} = \frac{8}{32} = \frac{1}{4}$$

$$k = \frac{W}{\Delta} = \frac{8}{2} = 4$$

$$\beta = 2$$

$$2\gamma = \frac{f}{m} = \frac{2}{1/4} = 8$$

$$\omega^2 = \frac{k}{m} = \frac{4}{1/4} = 16$$

$$\therefore x'' + 8x' + 16x = 0$$

$$\text{aux. eq. } m^2 + 8m + 16 = 0 \rightarrow m_1 = m_2 = -4$$

$$\therefore x(t) = C_1 e^{-4t} + C_2 t e^{-4t} = e^{-4t} (C_1 + C_2 t)$$

$$\therefore x(0) = 0, x'(0) = -3 \rightarrow C_1 = 0, C_2 = -3$$

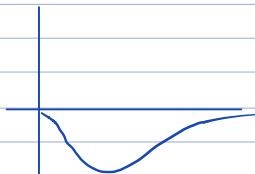
$$\therefore x(t) = -3t e^{-4t}$$

Extreme displacement: max or min of $x(t)$:

$$x'(t) = -3e^{-4t} + 12t e^{-4t} = 0 \quad (-) \rightarrow -3 + 12t = 0 \rightarrow t = \frac{1}{4}$$

$$\begin{aligned} x''(t) &= 12e^{-4t} + 12e^{-4t} - 48t e^{-4t} \\ &= 24e^{-4t} - 48t e^{-4t} > 0 \end{aligned}$$

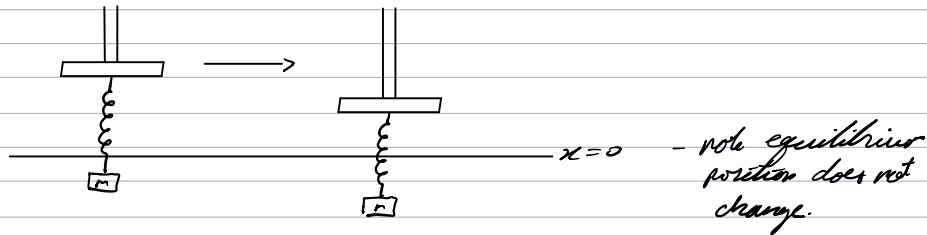
$$\therefore \text{Extreme disp. is } x\left(\frac{1}{4}\right) = -\frac{3}{4e}$$



5.1.3 Spring/Mass Systems: Driven Motion

Assumption: something drives the support with force $f(t)$

Driven motion:



DE of Driven Motion with Damping:

$$mx'' = -kx - \beta x' + f(t) \iff x'' + 2\lambda x + \omega^2 x = F(t)$$

$$2\lambda = \beta/m, \omega^2 = k/m, F(t) = f(t)/m$$

Method of solution: undetermined coefficients or variation of parameters

Recall the gen. sol. takes form

$$x(t) = x_c(t) + x_p(t).$$

Now call:

$x_c(t)$ = transient solution

$x_p(t)$ = steady-state solution

Ex

discretize and solve the IVP:

$$\frac{1}{5} x'' + 1.2 x' + 2x = 5 \cos(4t)$$

$$x(0) = \frac{1}{2}$$

$$x'(0) = 0$$

Compare to $mx'' + \beta x' + kx = f(t)$ (we use kg, m, but sleep, ft also work):

mass $m = \frac{1}{5}$ Kg

damping $\beta = 1.2$

spring constant $k = 2$

driving force $f(t) = 5 \cos(4t)$

mass released $\frac{1}{2}$ m from equilibrium at no initial velocity

Solution: Standard form: $x'' + 6x' + 10x = 25 \cos 4t$

$$1) \quad m^2 + 6m + 10 = 0 \implies m_1 = -3 + 2i, m_2 = -3 - 2i$$

$$\therefore x_c(t) = e^{-3t} (C_1 \cos t + C_2 \sin t)$$

$$\begin{aligned}
 2) \text{ Guess } x_p(t) &= A \cos 4t + B \sin 4t \\
 x_p'(t) &= -4A \sin 4t + 4B \cos 4t \\
 x_p''(t) &= -16A \cos 4t - 16B \sin 4t
 \end{aligned}$$

plug into $x'' + 6x' + 10x = 25 \cos 4t$ and find $A = \frac{-25}{102}$, $B = \frac{50}{51}$

$$3) x(t) = \underbrace{x_c(t)}_{\text{transient}} + \underbrace{x_p(t)}_{\text{steady state}}.$$

Now use initial conditions to find c_1, c_2 .

5.1.4 Series Circuit Analogue

Just a renaming of terms

$$x \rightarrow q:$$

$$L q'' + R q' + \frac{1}{C} q = E(t).$$

$$i(t) = q'(t)$$

$$\begin{array}{ccc}
 \text{Solution } q(t) = \underbrace{q_c(t)}_{\text{transient charge}} + \underbrace{q_p(t)}_{\text{steady-state charge}} & \rightarrow & i(t) = \underbrace{i_c(t)}_{\text{transient current}} + \underbrace{i_p(t)}_{\text{steady-state current}}
 \end{array}$$